

Patent

### Magnetic Latching Solenoid

#### Field of the Invention

The present invention relates to a solenoid construction, and in particular, to a magnetic latching solenoid.

#### Background of the Invention

Magnetically latched solenoid structures are well-known in the art, and have utilized various permanent magnet materials for latching purposes, i.e. wherein a magnet acts to retain an independently operable solenoid plunger adapted for linear motion of a plunger operated push and/or pull actuating rod for motivating electrical switchgear towards open and/or closed circuit position. Prior art devices have shown placement of a permanent magnet circuit inside the solenoid's magnetic circuit, and energizing the solenoid coil to cancel out the field of the permanent magnet, or to over power the magnetic field to affect motion. This materially affects the action of the operating components towards movement and latching activity.

#### Summary of the Invention

It is an object of the present invention to provide a magnetic latching solenoid design, which improves upon the prior art by locating the latching permanent magnet(s) assemblies externally of the solenoid operating

mechanism. This novel design approach outperforms the prior art in actuation speed and magnetic efficiency. The basic design concept is preferably used in connection with bi-directional operated latching solenoids. Certain 5 aspects of the magnetic latching concept disclosed herein have application in both single and dual directional solenoid structures.

It is another object of the invention to provide a magnetically operated actuator device, utilizing a 10 permanent magnet latching assembly incorporating high-energy, permanent magnets of rare earth or other relatively fragile permanent magnet materials, and to provide a mechanical structure that protects such materials from damaging impact when subjected to motion 15 of a solenoid plunger. The present concept may also use ceramic or Alnico magnets where their magnetic parameters permit.

Further, it is an object of the invention to provide a common pole piece in the center of the solenoid 20 assembly. This allows the two axially spaced solenoid portions to operate magnetically independently, unlike conventional dual action solenoids, which suffer from magnetic leakage around opposite ends of the unit. Further, the present concept provides for the oppositely 25 disposed latching members to operate independently from one another and from their respective solenoid construction.

Still another object of the invention is to meet industry requirements for circuit breakers controlled by 30 the present dual-action solenoid, which is: Trip-Close-Trip, all taking place on stored energy. The disclosed design can accomplish this function at a low energy level, thus increasing storage cost efficiency.

It will be apparent upon reading the following 35 description of the preferred embodiment that the

invention provides, in its bi-directional mode, three movable structures assembled in one housing, one of which structures has linkage to the work load. The magnetic latching structures are magnetically independent of the 5 solenoid structures, and each of the solenoids are magnetically independent of the other solenoid.

Brief Description of the Drawings

Further objects and advantages of this invention will become apparent from the following description taken 10 in conjunction with the accompanied drawings in which:

Figure 1 is a longitudinal sectional view, taken along lines 1-1 of Figure 2, of a bi-directional latching solenoid made in accordance with the teachings of the present invention.

15 Figure 2 is an end plan view of the bi-directional latching solenoid of Figure 1, and including a surrounding mounting support for the solenoid assembly.

20 Figure 3 is an exploded, perspective view of a permanent magnet latching subassembly, and in particular, a subassembly illustrating the components arranged for cooperation with a respective solenoid armature and ultimately act to magnetically latch the armature and solenoid push/pull rod in a desired operating position and in accordance with the teachings of this invention.

25 Figure 4 is a perspective view of the latching subassembly of Figure 3 and illustrating the components of the assembly in operating position relative to one another and with respect to a precision ground planar aligning surface shown in phantom view.

30 Description of the Preferred Embodiment

Like parts illustrated and described herein are designated by like reference characters.

Referring to the drawings, and particularly to Figure 1, there is illustrated a bi-directional version 35 of the magnetic actuator device, or solenoid 10, of the

present invention.

The bi-directional latching solenoid 10 preferably comprises a magnetic steel, tubular housing member 11. The housing 11 may be mounted to a vacuum bottle 5 interrupter, or the like, by means of mounting clamps 14 shown in further detail in Figure 2. The clamps 14 may be fastened in place by means of a bolt and nut fastener 15 inserted in aligned apertures (not shown) of laterally extending, oppositely disposed, bifurcated tang members 10 16. The tang members 16 are mounted for lateral support by extending cantilever plates 16a. Additional structural support may be obtained from a plurality 15 (four, in this case) of radially extending apertured ears 17. The apertures 18 in each of the ears 17 are provided to receive elongated supporting rods 19. The rods 19 are each positioned in radially spaced, coaxial alignment with the tubular housing 11 to provide longitudinal support for substantially the entire length of the magnetic actuator device 10. The preferably circular 20 inner clamping surface 12 of the respective clamps 14 ensures avoidance of ovality of the desired circular grooved outer surface of the tubular housing 11.

In the case of the presently described bi-directional solenoid apparatus 10, it is preferred to 25 provide individually operated, longitudinally spaced solenoid coil assemblies 20L and 20R. The coil assemblies 20L, 20R are respectively positioned and supported at opposite sides 21L, 21R of a centrally located stationary magnetic pole piece 22. The pole 30 piece 22 is secured in place by means of conventional retaining snap rings 23L and 23R located at the under-cut shoulder portions 24L and 24R located at opposite sides of the pole piece 22. Oppositely disposed non-magnetic tubular bobbins, or coil-supporting sleeves 27L and 27R 35 are each further provided within through-bore 26L, 26R

for slidably receiving and supporting respective armatures, or plungers 28L and 28R.

It will be noted that like parts are denoted in the drawings with like reference numerals, but with the additional indicia of "L" or "R" to indicate respective left and right locations as viewed with respect to the view of Figure 1. Accordingly, the cooperating components of the respective latching mechanisms are associated with the movement of the armature 28L responsive to current flowing through the coil 20L, and likewise with the cooperating components associated with the armature 28R and its operating coil 20R. The operations of the components of the respective latching mechanisms are the same, except for alternative direction of longitudinal movement of the armatures, or plungers 28L and 28R under the influence of their respective coils 20L or 20R. The solenoid coils 20L and 20R are preferably wound on non-magnetic, tubular bobbins 27L and 27R, respectively. In order to ensure positive alternative linear movement of the plungers 28L, 28R, the operating rod 46 and the clapper members 36L, 36R are each preferably threadingly (see threads 49) and adhesively (LOCTITE® 680) secured to the push/pull operating rod 46, and are further arranged to alternatively move the rod 46 in response to the electro-magnetic action of the respective solenoid coils 20L and 20R. The rod 46 is preferably threaded end-to-end to provide additional stability along its length.

As further illustrated in the view of Figure 1, the dual action, or bi-directional, solenoid structure 10 includes the aforementioned coils 20L and 20R, respectively wound to provide respective alternative, bi-directional, linear motion to magnetic plungers, or armatures, 28L and 28R. The common stationary pole piece 22 allows the two axially spaced solenoid assemblies to operate magnetically independently, and thereby

materially reduce magnetic leakage around the opposite ends to an insignificant level. The respective armatures or plungers 28L and 28R are arranged so that at the end of their respective strokes, they will abut the 5 respective sides 21L and 21R of the stationary pole piece 22 under the influence of a respective electromagnetic coil 20L or 20R. The axially spaced, plungers 28L and 28R are each preferably threadingly (see threads 49) and adhesively (LOCTITE® 680) secured to the push/pull 10 operating rod 46, and are further arranged to alternatively move the rod 46 in response to the electromagnetic action of the respective solenoid coils 20L and 20R.

As will hereinafter be discussed, the spring 32L is 15 "lighter" than the "heavier" spring 32R. That is, the spring 32R for this particular solenoid configuration is preferably wound from 0.135" stainless steel type 302 wire with 2.94 active coils, and the lighter spring 32L is preferably wound from 0.095" stainless steel type 302 20 wire with 2.99 coils providing a spring rate of 3.33 pounds per inch. The heavy spring 32R provides a spring rate of 22.01 pounds per inch.

The inner volutes 34L and 34R of the springs 32L, 32R, respectively, rest against the inwardly facing 25 recessed surfaces 35L and 35R of magnetic coupling members, exemplified herein by the plunger clapper members 36L and 36R.

It will be observed, as viewed in Figure 1, that the bi-directional solenoid 10 includes independently left 30 and right operable, magnetically latching mechanisms, which are located at opposite ends of the tubular housing 11. The axial spacing is insured by means of c-shaped snap rings 71L and 71R ended by conventional, magnetic flux washers 77L and 77R. The tubular bobbins 27L and 35 27R complete the physical assembly. Again, directing

attention to Figure 1, it will be observed that the left-hand magnetic latching assembly is axially spaced from the solenoid assembly comprising the coil 20L wound on the bobbin 27L, and its respective armature or plunger 28L. The right-hand magnetic latching assembly is also axially spaced from the solenoid assembly comprising the coil 20R wound on the tubular bobbin 27R and its respective armature or plunger 28R and located at the right of the snap ring 71R.

The outer volutes 38L and 38R of the respective biasing coil springs 32L and 32R are seated within inwardly facing re-entrant counter bores 48L and 48R formed on the inwardly facing surfaces of outer magnet holders 50L and 50R. The outer magnet holders 50L and 50R are restrained from outward longitudinal movement with respect to the tubular housing 11 by means of conventional snap rings 51L and 51R located at opposite ends of the housing 11. However, it is preferred to provide a narrow mechanical gap 89 between the respective outer magnetic holders 50L and 50R and the shoulders 90L and 90R. Thus, the gap 89 will permit enough axial "play" during the impacting motion of a respective plunger 28L, 28R. As will be later discussed, magnetic gap 88 will be narrowed to almost zero for optimal magnetic latching attraction of the mating components.

Operation of the device will next be described in connection with the view of Figure 1, and assuming the left side of the device 10 is shown in the left side latched position. Upon energizing the coil 20L, the solenoid force builds until it overpowers the force created by the latched magnets 65L and the magnetic coupling member, or clapper 36L. It does not drive the flux of the magnets as is done in many prior art devices. The plunger or armature 28L will be rapidly accelerated towards the pole piece 22. Meanwhile, during the motion

of the plunger 28R, and just before impact, the bias spring 32R will act to momentarily keep the sensitive magnet structure, including the respective magnet discs 65R, out of the way, i.e. being isolated from direct contact with members that will be impacted, until such time after the plunger 28L impacts upon the side 21L of the pole piece 22. At this time, the magnets 65R which are of sufficient strength to overcome the bias of the spring 32R, and the magnetic reluctance of the air gap 88, and will pull themselves up to the plunger clapper 36R to a latched condition. The like components are illustrated in latched position at the left side of the housing 11. The relationship of the cooperation components will complete a virtually closed magnetic circuit. The disclosed and preferred magnetic coupling of cooperating magnetic components provides a relatively large magnetic force. The forces build up to the large magnetic forces exerted by the selected permanent magnetic discs 65R and the almost zero air gap 88 resulting from the very tight tolerances of mating components of the preferred configuration. The average velocity of test devices has been found to be about one (1) meter per second. Obviously, because of using substantially identical components and characteristics, similar results are obtained from the operating action of coil 20R upon its armature, or plunger 28R, but in the opposite direction. The actual speed depends on the load curves of the device being actuated.

It is also within the province of this invention to extend the concept of the biasing means to include the concept of entrapping and compressing air within sealed chambers 85L and 85R created between the outer magnetic holders 50L and 50R and their respective clapper members 36L and 36R.

35 It will be apparent that the left-side armature 28L

continues in motion until seating adjacent the pole piece 22 as shown in Figure 1. Again, with reference to Figure 1, during the alternative directional motion to the left, the opposite magnet assembly pulls toward and latches on to its plunger clapper or magnetic coupling member 36L, while overpowering the bias of the biasing spring 32L, which had kept the magnet assembly out of the way during the impact caused by the plunger seating motion. The high latching forces are obtained by optimizing the 10 surface areas of the mating components. The surface areas are designed to cause the highest magnetic flux densities through the completed magnetic circuit.

With further reference to the views of Figures 3 and 4, it will be observed that the components of each of the 15 independent magnetic latching mechanisms are preferably pre-assembled as an integral unit, as shown herein with the left-hand indicia "L". The integral units respectively comprise inner magnet holder 62L, 62R each of magnetic material arranged for inner surface support 20 of a pre-selected number of magnetic discs 65L, 65R, respectively. The outer surface of each of the magnetic discs 65L, 65R, are further retained by means of a middle magnet holder 67L, 67R. The magnet subassembly is held together by means of the threaded bore 70L, 70R, of an outer magnet holder 50L, 50R and the mating external threads 73L, 73R of the respective middle magnet holder 25 67L, 67R. The threaded areas are also coated with an adhesive such as LOCTITE® 680, and the entire assembly is held in compression by means of a non-magnetic threaded bolt 74L, 74R, the threads of which engage the threads 30 75L, 75R of the bore of the middle magnet holder 67L, 67R, in addition to a coating of an adhesive such as LOCTITE® 680. The flanged head 78L of the bolt 74L rests against the underside of the inner magnet holder 62L to 35 complete the subassembly. With reference to Figure 4, it

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will be noted that during assembly of the various cooperating parts, the parts are maintained in precise alignment by means of resting the inner surfaces 72L, 72R of the outer magnet holder 50L, 50R, and the innermost 5 holder 62L, 62R on the precision ground surface 80 of a conventional fixturing jig 81 (shown here in phantom). While this is the preferred means for holding the magnet subassembly together, it is to be understood and appreciated that the subassembly could be held together 10 utilizing an adhesive, a press-fit arrangement, an insert mold process or any other suitable means.

The magnetic discs 65L, 65R are preferably of a rare earth material exhibiting high magnetic energy per unit volume. A very satisfactory magnetic disc material may 15 be formed and fired from a commercially available material identified as "RMND114 GRADE 30 ROCHESTER". Since magnetic discs 65L and 65R made from this material, like all rare earth magnetic materials, are relatively fragile, the operating elements of the present invention 20 protects them against relatively rough and abrupt operation of the alternative motion of the armatures or plungers 28L, 28R. In particular, the present concept provides a means of isolating the magnets from the shock 25 of impact of the respective plunger 28L, 28R at the end of travel and abutment against a respective surface 21L or 21R of the stationary pole piece 22.

It is also to be observed that each of the magnetic discs 65L, 65R have the same magnetic orientation. That 30 is, each of their respective North and South poles face in the same direction. With this arrangement, the overall magnetic attraction will be enhanced. And also of importance, the magnets will be physically oriented with their respective North and South poles each facing the same direction. Assembly will require preventing the 35 repulsion of adjacent magnets.

With reference to Figure 1, it will be noted that in the present case, the axial lengths of the respective magnetic discs 65L are deliberately pre-selected to be less than the respective axial lengths of the discs 65R. 5 The total axial lengths of the respective discs 65L combined with the axial length of the inner most holder 62L is identical with the total combined axial lengths of discs 65R and their respective innermost magnet holder 62R. Thus, dimensions of the various magnetic latching 10 components may be varied to provide the respective dimensional gaps 88 of the left hand and right hand magnetic latching subassemblies.

In the disclosed preferred embodiment of the dual latching solenoid assembly 10, which may operate a 15 conventional vacuum bottle circuit breaker, it has been determined that a satisfactory magnetic structure may utilize an 8/4 magnetic construction. That is, the right-hand latching magnet assembly preferably comprises eight (8) magnetic discs 65R, along with the 20 aforementioned heavier biasing spring 32R, whereas four (4) magnetic discs 65L utilize the combination of the four (4) discs 65L with the lighter biasing spring 32L.

The preferred design allows the use of multiple, 25 low-cost, readily available magnets 65L and 65R, instead of a single conventional, high-cost, custom-made, toroidal magnets. A single, or even stacked toroidal magnet, do not provide the cost effectiveness achieved by the arrangement of individually magnetic discs 65L, 65R, which are preferred in the assembly exemplified by the 30 views of Figure 3 and Figure 4.

It will be further apparent that the present invention includes three movable structures assembled in one housing, one of which has linkage to the workload. The latching structures are magnetically independent of 35 the solenoid structures, and each solenoid is

magnetically independent of the other solenoid. Also, the latching structures are not affected by the impacting of the solenoid structures. The biasing means, in the form of springs 32L and 32R keep the latching structure 5 out of the way until the impact of the respective plunger with its side of stationary pole piece 22 has occurred. After the pull force of the latching structure, even with a relatively large air gap, is strong enough to compress the respective bias spring 32L or 32R, and to finally 10 seat on the plunger coupling member, or clapper 36L or 36R. Once seated, the resulting air gap 88 is almost zero, and high latching force can thus be obtained. In addition, high actuation speed is possible, since no 15 solenoid motion begins until the solenoid force exceeds the latching structure force.

The design further allows the use of multiple, low cost, readily available magnets 65L or 65R, instead of one high-cost custom magnet.

It will be observed that the construction of the 20 latching assembly substantially cancels out the "stack up" of machining tolerances, thus making the device cost effective.

It will be further observed that the bi-directional 25 magnetic latching solenoid 10 illustrated and described herein will provide a convenient and easily assembled and operated dual unit. It will be apparent that the unit may utilize substantially identical magnetic latching components for a single directionally operated solenoid by simply utilizing the respective latching components of 30 either the right hand or the left hand component assemblies of the view of Figure 1.

It will also be apparent that the herein disclosed 35 configuration of the latching solenoid construction may further contemplate a magnetic configuration, or arrangement, which includes a polar array of two or more

equally spaced disc magnets, two or more magnetic arcuate sections, or a single toroidal magnet of pre-selected magnetic strength.

5 The foregoing is considered as illustrative only of the principles of the invention. Furthermore, since numerous modifications and changes will readily occur to those skilled in the art, it is not desired to limit the invention to the exact construction and operation shown and described. While the preferred embodiment has been  
10 described, the details may be changed without departing from the invention, which is defined by the claims.